



*Original Article*

## Comparison of automatically generated reaction mechanism for oxidation of simple hydrocarbons in IC engine

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Received 20 August 2009; Accepted 28 September 2011

### Abstract

In this work, a detailed kinetic reaction mechanism, consisting of 208 reactions and 79 species, has been developed to describe the oxidation of simple hydrocarbon fuel (natural gas) in IC engine. The performance of the proposed mechanism is tested using simulation, tool CHEMKIN 4.1.1, and experimental measurements. The simulation results of the proposed reaction scheme were compared with those of reference mechanisms (GRI v3.0 and Konnov 0.5 version) as well as experimental data. Based upon simulation results, it can be concluded that the proposed mechanism shows good concordances with GRI 3.0 mechanism especially in the prediction of temperature, pressure, and major product species ( $H_2O$ ,  $CO_2$ ) profiles at stoichiometric conditions ( $\phi = 1.0$ ). Although, there are some discrepancies among each predicted profile, the proposed detailed mechanism is good to describe the oxidation of natural gas in IC engine. The experimental data also showed favorable results for prediction of major product species ( $CO_2$ ,  $H_2O$  &  $CO$ ) at various engine operating speeds in idle mode.

**Keywords:** IC Engine, mechanism, simulation, stoichiometric conditions

### 1. Introduction

The understanding of oxidation kinetics of simple hydrocarbons such as methane ( $CH_4$ ), ethane ( $C_2H_6$ ), propane ( $C_3H_8$ ), and mixture of these (natural gas) is a key objective of combustion chemistry. To achieve this understanding, a detailed mechanism is required. The detailed mechanisms involved hundreds of chemical species and thousands of reaction steps. Detailed combustion mechanisms are based on measurements of elementary gas phase reaction coefficients. The measurements have provided the temperature and pressure dependencies of the overall rate coefficient. The mechanisms of Warnatz and Walker (Warnatz, 1983, 1993; Walker *et al.*, 1997) have been used by many people, but were mainly created as illustrations in combustion modeling papers. Another widely used mechanism is the Miller-

Bowman NO mechanism (Miller and Bowman, 1989), which, like the Warnatz mechanism, is now out of date in many respects. The GRI mechanism (Gregory *et al.*, 2011) is used for study of oxidation of methane. This mechanism is based on elementary reactions, where combinations of experimental and theoretically determined values are assigned to the rate parameters. One of the pioneering ideas of the GRI mechanism was that the mechanism, its testing, and related information are available through Internet ([http://www.me.berkeley.edu/gri\\_mech/](http://www.me.berkeley.edu/gri_mech/)). Other combustion mechanisms also available through Internet were developed by Konnov (2000) and the Leed's mechanisms (<http://www.chem.leeds.ac.uk/Combustion/Combustion.html>). These mechanisms were used to understand the various combustion phenomenon like flame speed, oxidation of fuel mixture of simple hydrocarbon, flame structure, etc.

EXGAS is an automatic tool for kinetic reaction mechanism for hydrocarbon fuels (developed at Nancy, France). It also generates allied data (thermo chemical) automatically using its associated components, i.e., THERGAS

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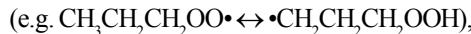
and KINGAS (Muller *et al.*, 1995; Glaude *et al.*, 2000). EXGAS was utilized for the generation of oxidation mechanism for mixture of species (Come *et al.*, 1996; Glaude *et al.*, 1997; Glaude *et al.*, 1998; Warth *et al.*, 1998; Barbé *et al.*, 1995). The reacting mixture species have special 1D notation. The software has built-in-notations for methane and ethane species as base species. These one dimensional (1D) notations were linked by the computer to an internal representation, which allows species reactants and products to be stored in a unique format using a canonicity algorithm. A detailed primary mechanism by EXGAS contains the following types of reactions primary required to understand the oxidation mechanism of hydrocarbons (Battin *et al.*, 2004);

- 1) Unimolecular and bimolecular initiation steps  
(e.g.  $\text{C}_3\text{H}_8 \leftrightarrow \cdot\text{CH}_3 + \cdot\text{C}_2\text{H}_5$ ;  $\text{C}_3\text{H}_8 + \text{O}_2 \leftrightarrow \cdot\text{C}_3\text{H}_7 + \cdot\text{OOH}$ ),
- 2) Reactions leading to alkenes from alkyl and hydroperoxyalkyl radicals
  - Decompositions by beta-scission  
(e.g.  $\cdot\text{C}_3\text{H}_7 \leftrightarrow \cdot\text{CH}_3 + \text{C}_2\text{H}_4$ ),
  - Oxidations to form the conjugated alkene and  
 $\cdot\text{OOH}$   
(e.g.  $\cdot\text{C}_3\text{H}_7 + \text{O}_2 \leftrightarrow \text{C}_3\text{H}_6 + \cdot\text{OOH}$ ),

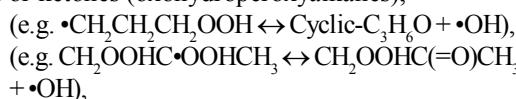
3) Additions of alkyl and hydroperoxyalkyl radicals to a molecule of oxygen



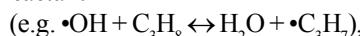
4) Isomerizations of alkyl and peroxy radicals involving cyclic transition state



5) Decompositions of hydroperoxyalkyl and dihydroperoxy alkyl radicals to form cyclic ethers, alkenes, aldehydes or ketones (oxohydroperoxyalkanes),



6) Metathesis reactions to abstract an H atom from the initial reactant



7) Termination steps:

- Combinations of two free radicals  
(e.g.  $\text{CH}_3 \cdot + \text{CH}_3 \cdot \leftrightarrow \text{C}_2\text{H}_6$ ),
- Disproportionations of peroxy radicals with  
 $\cdot\text{OOH}$   
(e.g.  $\text{C}_3\text{H}_7\text{OO} \cdot + \cdot\text{OOH} \leftrightarrow \text{C}_3\text{H}_7\text{OOH} + \text{O}_2$ ).

## 2. Materials and Methods

### 2.1 Generation of detailed primary kinetic mechanism by EXGAS

We generated kinetic mechanisms of simple hydrocarbon fuel (natural gas; a mixture of methane, ethane, propane, and butane) combustion in an internal combustion engine using EXGAS software. It generates the reaction scheme automatically and has built in 1D notations for

molecules of methane, ethane in reaction base, and we input the formulas in form of  $\text{ch3/ch2/ch3}$ ,  $\text{ch3/ch2/ch2/ch3}$ , and  $\text{//(o)2}$  for propane, butane, and oxygen molecule, respectively. Initially, the software generated a mechanism with 502 reaction steps and this mechanism has been reduced automatically to 69 reaction pathways by chemical lumping technique. Thermodynamics data (JANAF format) necessary for CHEMKIN is produced by THERGAS (built-in-module in EXGAS) automatically. This model contains 60 reactions of various types such as initiation, beta-scission, oxidation, branching, termination, and others, and has 79 species. The reaction is further simplified by chemical lumping method and about 33% reaction steps are reduced.

### 2.2 Coupling of EXGAS mechanism and leeds $\text{NO}_x$ mechanism

The reaction by EXGAS was coupled with Leeds  $\text{NO}_x$  mechanism (version 2.0 available at <http://www.chem.leeds.ac.uk/Combustion/nox.htm>) using a software OPENCHEM WORKBENCH ([http://openchemwb.sourceforge.net/OpenChemWB\\_Home.html](http://openchemwb.sourceforge.net/OpenChemWB_Home.html)). The purpose of this coupling was to develop a comprehensive reaction scheme (model) that describes  $\text{CH}_4/\text{NO}_x$  chemistry for oxidation of simple hydrocarbon fuel (natural gas) at high temperature (above 2,000 K).

The major objective of this study is to develop a reaction kinetic model, which is capable of predicting the formation of major gaseous pollutants from engine exhaust. The results were evaluated by comparing with profiles of in-cylinder temperatures, pressures, major species, and pollutant emissions (only  $\text{CO}$ ,  $\text{NO}_x$  as  $\text{NO}$  and  $\text{NO}_2$  and  $\text{NH}_3$ ) of GRI 3.0 and that of A. Konnov's detailed reaction (Konnov, 2000) for small hydrocarbons.

### 2.3 General simulation inputs

Inputs of the simulation are

- Chemical reaction mechanisms with Arrhenius coefficients.
- Thermodynamic and transport data.
- Mole fraction of reactive species (in our case of natural gas:  $\text{CH}_4$ ,  $\text{C}_2\text{H}_6$ ,  $\text{C}_3\text{H}_8$ ,  $\text{N}_2$ ,  $\text{CO}_2$ , Air- $\text{O}_2$ ) for various equivalence ratio and geometrical parameters of ICE.

Through these inputs, we can extract information from simulation such as

- Temperature profiles
- Pressure profiles
- Main species profiles, and
- Total rate-of-production.

## 3. Results and Discussions

The performance of this newly proposed (developed by coupling EXGAS mechanism and Leed's  $\text{NO}_x$  mechanism version 2.0) kinetic mechanism was investigated using Internal Combustion Engine (ICE) module of CHEMKIN 4.1.1

Table 1. Menu for generation of primary mechanisms by EXGAS.

Types of Primary Reactions	Options Selected for Generation of Primary Reaction Mechanism	
	Yes/No	No. of Reactions
<i>Unimolecular initiations</i>	Yes	3
<i>Bimolecular initiations</i>	Yes	3
<i>Additions with oxygen</i>	No	0
<i>Isomerizations</i>	No	0
<i>Beta-scissions</i>	Yes	7
<i>Decompositions to o-rings</i>	No	0
<i>Oxidations</i>	Yes	3
<i>Branching</i>	No	0
<i>Metatheses</i>	Yes	43
<i>Combinations</i>	Yes	10
<i>Dismutations</i>	No	0
Number of Reactions in Lumped Primary Mechanism		69
Total Number of reactions (after Coupling with Lead's NO <sub>x</sub> Mechanism)		208
No of Species		79
Number of primary molecules		23
Number of free radicals		22
Number of lumped molecules		34

(Kee *et al.*, 2007). The combustion of natural gas mixture was simulated in an engine cylinder with the proposed kinetic reaction scheme (Table 2). In this simulation, various engine parameters (crank rotary speed, compression ratio, cylinder displacement volume, etc.) and initial inlet gas composition were utilized (Table 3). The other necessary simulation input data are thermodynamic data, which for proposed kinetic reaction scheme was automatically generated by THERGAS module of EXGAS software. The simulation with the proposed kinetic model produces about 46 species in the product mixture, which are purely dependent on the reacting and product species in the pathway reaction steps.

Figure 1 shows the simulation results of molar conversion of inlet natural gas mixture and oxidizer for the three mechanisms. The simulation is carried out at stoichiometric condition ( $\phi=1.0$ ) at  $P=1.0$  atm and  $T=2,000^{\circ}\text{C}$ . During the simulation, it was observed that the maximum (nearly 100%) conversion of reacting species could be achieved under fuel rich conditions (i.e.  $\phi>1.0$ ). The major fuel components (i.e. CH<sub>4</sub> and C<sub>2</sub>H<sub>6</sub>) show nearly complete conversion to products at an equivalence ratio of  $\phi\geq 1.3$  (at rich conditions). The profile of propane conversion indicates that complete conversion could be achieved at an equivalence ratio of far greater than 1.3. It can be concluded that maximum conversion of natural gas occurs under rich conditions ( $\phi>1.0$ ).

Figure 2 shows that the pressure and temperature profiles obtained with the three detailed kinetic models for adiabatic and stoichiometric condition ( $T_{ini}=500^{\circ}\text{C}$ ,  $P_{ini}=1.0$  atm and  $\phi=1.0$ ). The reference mechanisms (GRI version 3.0

and Konnov version 0.5) predict the maximum engine pressure of about 40.7 atm, while the proposed mechanism predicts the  $P_{max}$  of nearly 39.4 atm. With the proposed kinetic mechanism, the predicted pressure is about 3% lower than the reference mechanisms (GRI 3.0 and Konnov 0.5). As shown in this figure, there is some discrepancy in the temperature profiles between the proposed mechanism and both referenced detailed mechanisms. The proposed mechanism shows a later start of combustion and lower engine cylinder peak temperature (about 234 K).

Figure 3 illustrates the fuel species (natural gas components; CH<sub>4</sub>, C<sub>2</sub>H<sub>6</sub>, C<sub>3</sub>H<sub>8</sub>) and oxidizer (air-O<sub>2</sub>) profiles for the adiabatic and stoichiometric conditions. The proposed mechanism shows closer consumption pattern with GRI 3.0 mechanism especially for methane (CH<sub>4</sub>) and atmospheric oxidizer (O<sub>2</sub>) profiles under the similar simulation conditions. All species profiles indicate that the proposed mechanism exhibits complete combustion of natural gas in the IC engine and depicts that the reactions proposed represents the combustion of the natural gas. Figure 4 illustrates the two major combustion product species (CO<sub>2</sub> and H<sub>2</sub>O). This figure also shows that Konnov 0.5 mechanism exhibits some earlier start of combustion and later end of the combustion in H<sub>2</sub>O profiles. The proposed detailed mechanism and GRI 3.0 mechanism exhibit similar combustion behaviors, but during the ending of the combustion process, the proposed mechanism illustrates the earlier completion of the combustion. There is some discrepancy in the species profiles near the completion of combustion. The proposed mechanism predicts

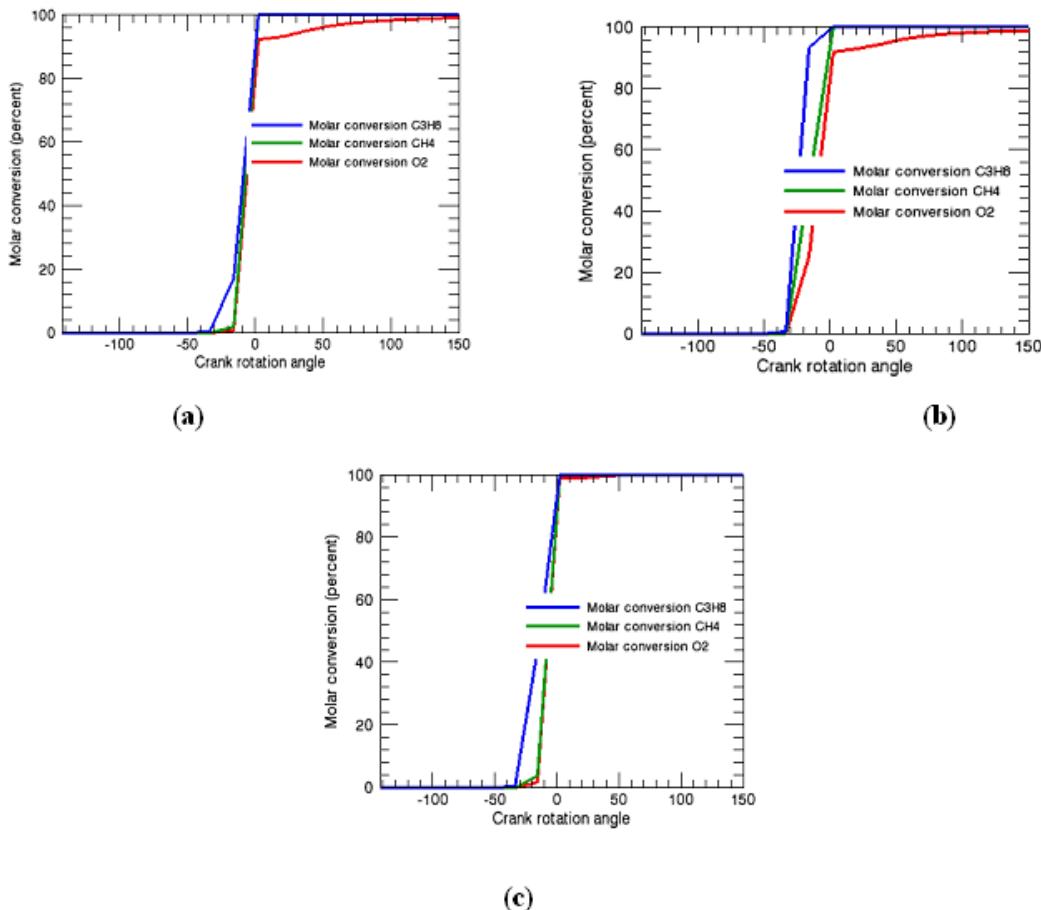
Table 2. Pressure dependent reactions in the proposed kinetic reaction mechanism for oxidation of simple hydrocarbon fuels at high temperature (above 2,000 K).

Sr. No	Reactions Considered	k = A T <sup>**b</sup> exp(-E/RT)		
		A	b	E
1	$C_3H_8 + R_1H \Rightarrow H_2 + R_{21}C_3H_7$	9.00E+06	2	5000
2	$C_4H_{10} + R_1H \Rightarrow H_2 + R_{20}C_4H_9$	5.70E+07	2	7700
3	$C_3H_8 + R_2OH \Rightarrow H_2O + R_{21}C_3H_7$	2.60E+06	2	-765
4	$C_4H_{10} + R_2OH \Rightarrow H_2O + R_{20}C_4H_9$	5.40E+06	2	450
5	$C_4H_{10} + R_4CH_3 \Rightarrow CH_4 + R_{20}C_4H_9$	6.00E-01	4	8200
6	$C_3H_8 + R_5CHO \Rightarrow HCHO + R_{21}C_3H_7$	1.00E+07	1.9	17000
7	$C_4H_{10} + R_5CHO \Rightarrow HCHO + R_{20}C_4H_9$	2.00E+05	2.5	18500
8	$C_3H_8 + R_6CH_2OH \Rightarrow CH_3OH + R_{21}C_3H_7$	6.00E+01	3	12000
9	$C_4H_{10} + R_6CH_2OH \Rightarrow CH_3OH + R_{20}C_4H_9$	2.00E+02	3	14000
10	$C_3H_8 + R_{21}C_3H_7 \Rightarrow C_3H_8 + R_{19}C_3H_7$	8.40E-03	4.2	8700
11	$C_4H_{10} + R_{21}C_3H_7 \Rightarrow C_3H_8 + R_{22}C_4H_9$	5.60E-03	4.2	8000
12	$H_2 + CN \Rightarrow HCN + H$	1.93E+04	2.9	6.8
13	$CH_4 + CN \Rightarrow HCN + CH_3$	9.03E+04	2.6	-1.2
14	$O_2 + N \Rightarrow NO + O$	9.03E+09	1	27.2
15	$O_2 + NH_2 \Rightarrow HNO + OH$	1.51E+12	-0.4	151
16	$O_2 + NH_2 \Rightarrow H_2NO + O$	1.10E+18	-1.3	140.6
17	$NO_2 + NH \Rightarrow HNO + NO$	1.00E+11	0.5	16.6
18	$NO_2 + NH_2 \Rightarrow N_2O + H_2O$	2.03E+17	-1.7	0
19	$NH_3 + H \Rightarrow NH_2 + H_2$	5.42E+05	2.4	41.5
20	$NH_3 + NH_2 \Rightarrow N_2H_3 + H_2$	7.94E+11	0.5	90.2
21	$N_2H_2 + O \Rightarrow NNH + OH$	1.00E+11	0.5	0
22	$N_2H_2 + NH_2 \Rightarrow NH + N_2H_3$	1.00E+11	0.5	141.3
23	$HCN + O \Rightarrow NCO + H$	8.45E+05	2.1	25.6
24	$HCN + O \Rightarrow NH + CO$	3.19E+05	2.1	25.6
25	$HCN + O \Rightarrow CN + OH$	2.22E+05	2.1	25.6
26	$HCN + OH \Rightarrow HO + CN + H$	5.85E+04	2.4	52.3
27	$HCN + OH \Rightarrow H + CO + NH$	1.98E-03	4	4.2
28	$HCN + CN \Rightarrow C_2N_2 + H$	3.80E+07	1.6	0.4
29	$HNCO + H \Rightarrow NCO + H_2$	2.05E+14	-0.3	84.7
30	$O + NH_2 \Rightarrow NH + OH$	6.90E+11	0.3	-0.8
31	$O + NH_2 \Rightarrow HNO + H$	8.93E+14	-0.5	1.4
32	$O + NNH \Rightarrow NH + NO$	1.65E+14	-0.2	-4.2
33	$O + HNO \Rightarrow OH + NO$	5.01E+11	0.5	8.3
34	$OH + NH \Rightarrow HNO + H$	1.00E+12	0.5	8.3
35	$OH + NH \Rightarrow N + H_2O$	5.01E+11	0.5	8.3
36	$OH + NH_2 \Rightarrow O + NH_3$	1.99E+10	0.4	2.1
37	$OH + NH_2 \Rightarrow NH + H_2O$	5.01E+11	0.5	8.3
38	$N + NH \Rightarrow N_2 + H$	6.31E+11	0.5	0
39	$N + NCO \Rightarrow NO + CN$	2.77E+18	-1	72.2
40	$NH + NH_2 \Rightarrow N_2H_2 + H$	1.51E+15	-0.5	0
41	$NH + NNH \Rightarrow N_2 + NH_2$	2.00E+11	0.5	8.3
42	$NH_2 + M \Rightarrow NH + H + M$	7.91E+23	-2	382.4
43	$NH_2 + HNO \Rightarrow NH_3 + NO$	5.01E+11	0.5	4.2
44	$NNH + M \Rightarrow N_2 + H + M$	2.50E+13	0.5	12.8

Note: Formulas with notation R in above reactions means they are radicals and obtained in the primary mechanism. For example;  $R_1H$  is a hydrogen radical,  $R_{21}C_3H_7$  is 1-propyle radical and  $R_{20}C_4H_9$  is the 1-butyle radical etc. M is third body component and formulas with  $Z$  denotes the alkene leading by metathesis to an allylic radical which can decompose by breaking a (C-C) bond

Table 3. Simulation inputs for combustion analysis.

Test Engine Specification (Installed in 4-Stroke 3-wheeler Automobile)		Initial Reactants (Natural Gas) Composition	
Parameters	Values	Component	Mole Fraction
Compression Ratio	10.0	CH <sub>4</sub>	0.888
Engine Displaced Volume (cm <sup>3</sup> )	63	C <sub>2</sub> H <sub>6</sub>	0.012
Engine Speed (rpm)	5000	C <sub>3</sub> H <sub>8</sub>	0.003
Connecting Rod to Crank Radius Ratio	1.632	CO <sub>2</sub>	0.026
Cylinder Bore Diameter (cm)	63	N <sub>2</sub>	0.071

Figure 1. Comparison of Molar Conversion of Inlet Gas Mixture at  $P_{in}=1.0$  atm,  $T_{in}=2,000^{\circ}\text{C}$  and  $\varphi=1.0$  for mechanism (a) GRI 3.0, (b) Konnov 0.5 and (c) proposed Mechanism.

the lower species fractions. During the expansion and exhaust strokes, profiles show the occurrence of equilibrium in the product species of CO<sub>2</sub> and H<sub>2</sub>O. The product species profile predict that the proposed mechanism exhibit the closer combustion cycle behavior with the GRI 3.0 and predicts lower fraction of the species than both reference mechanisms (GRI 3.0 and Konnov 0.5). Each mechanism dictates that CO<sub>2</sub> and H<sub>2</sub>O are formed as rapidly as fuel and oxygen are consumed. The deviation in reactants and products profiles

occurs because of the number of reaction steps and intermediates in each mechanism. Although there is a discrepancy in the species profiles, the proposed mechanism produces acceptable agreement with GRI 3.0 mechanism (reference detailed mechanism).

Figure 5 compares the prediction of emissions of major pollutant species (CO, NH<sub>3</sub>, NO, NO<sub>2</sub>, etc.) for each mechanism. The proposed mechanism produces lower fraction of CO than both GRI 3.0 and Konnov 0.5 mechanism.

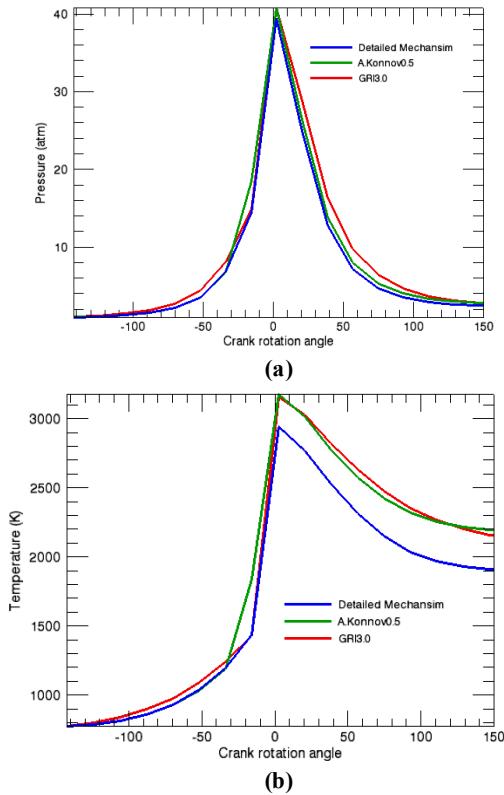


Figure 2. Predicted pressure and temperature profiles for  $P_{ini} = 1.0$  atm  $T_{ini} = 500^\circ\text{C}$  and Equivalence Ratio  $\varphi = 1.0$ .

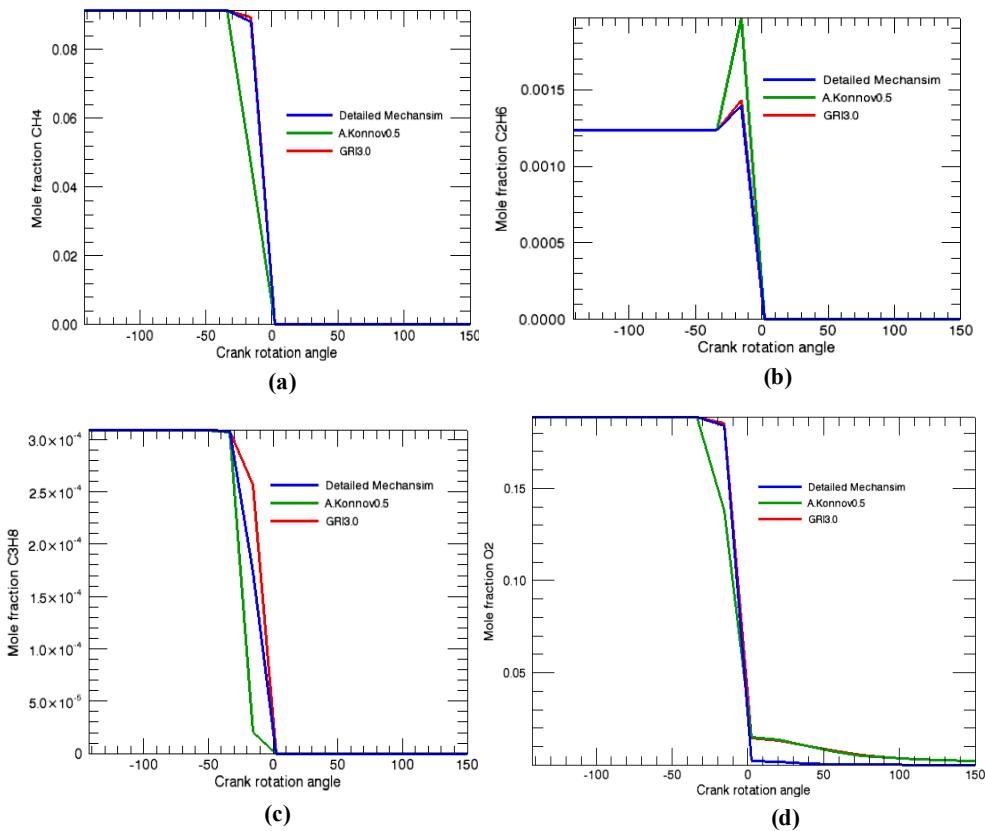
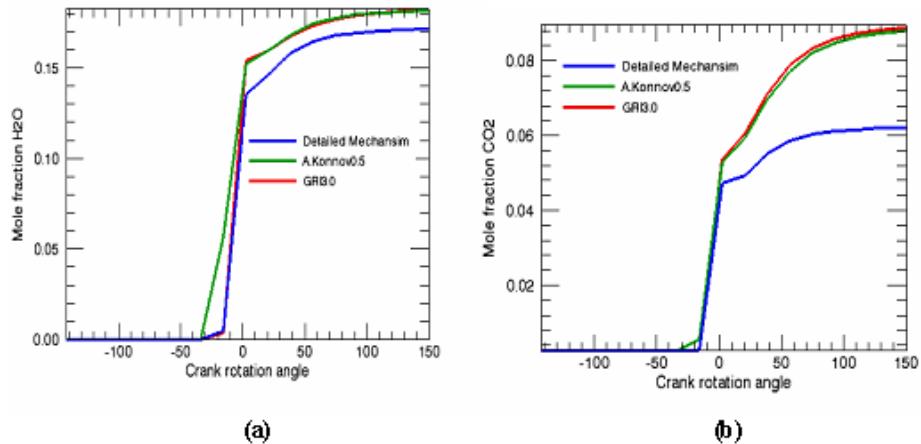
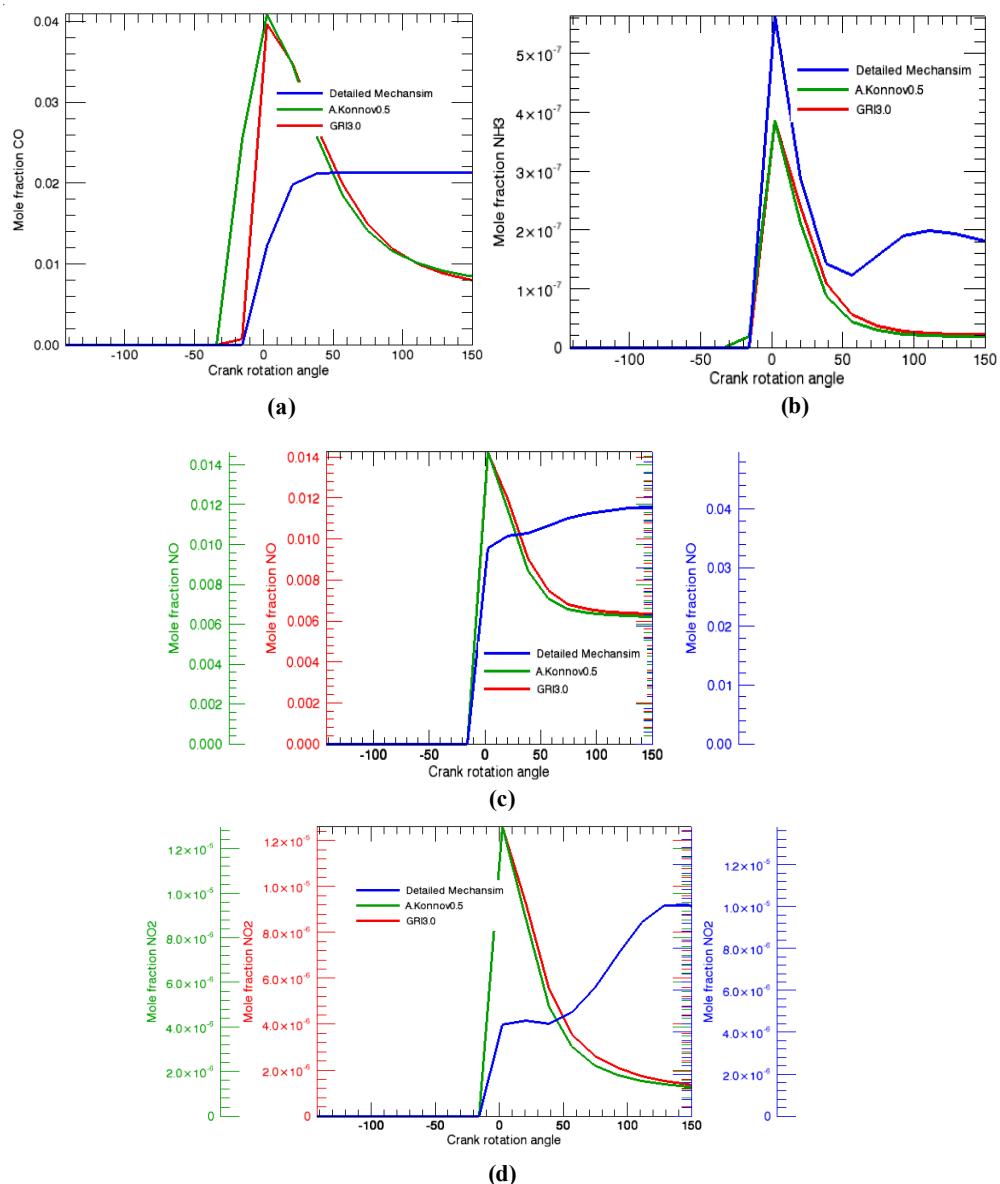


Figure 3. Comparison of fuel species and oxidizer profiles for adiabatic and stoichiometric conditions ( $P_{ini} = 1.0$  atm  $T_{ini} = 500^\circ\text{C}$  and equivalence ratio  $\varphi = 1.0$ ).

Figure 4. Major combustion product species profiles for equivalence ratio of  $\varphi = 1.0$ .Figure 5. Pollutants formation (CO, NH<sub>3</sub>, NO, and NO<sub>2</sub>) profiles for stoichiometric conditions ( $\varphi = 1.0$ ).

The predicted NO and NO<sub>2</sub> profiles show the formation after the combustion of fuel by the proposed mechanism and deviate significantly from those of the reference mechanisms. On whole, the mean fractions of each pollutant for the proposed mechanism is overpredicted but peak fraction are lower than the reference detailed mechanism (GRI 3.0 and Konnov 0.5). This could be due to the fact that in the proposed mechanism CO reacts by more multitude pathways than GRI 3.0 and Konnov 0.5 mechanisms.

In our experimental setup, exhaust composition and pressure were measured. The cylinder pressure was measured directly by the using a pressure transducer (Kistler, Model 603B1) combined with a charge amplifier (Kistler, Model 5004). Pressure data was recorded using NUDAC-TK version 2.0, a program for data acquisition and processing. The original data files contained cylinder pressure at crank angles 0°–720°. Each of three large files, about 969 mega bites each, contained about 9,000 combustion cycles. The engine speed, air/fuel ratio, and throttle setting were all held constant throughout the data collection period. The collected pressure data was plotted at various engine speeds. During pressure measurements, intake air pressure had a value of 38 kPa. The data for major species like CO<sub>2</sub>, CO, H<sub>2</sub>O were measured using online emission analyzer (Testo 350XL Model).

From the experimental data, the fraction of exhaust species (CO<sub>2</sub>, H<sub>2</sub>O, and CO) is determined at various operating engine speeds (Figure 6). Each plot show close agree-

ment between measured data and simulation results obtained by CHEMKIN 4.1.1 for proposed mechanism. The simulation predicts the closer cylinder pressure at lower engine speed.

Correlation coefficient of simulation data of each of three mechanisms with the experimental data was calculated. The simulation data of proposed mechanism shows a strong relation with experimental results for major combustion products (CO<sub>2</sub>, H<sub>2</sub>O, and CO) and resulting values of correlation coefficients were 0.9353, 0.9776, and 0.9756, respectively, as shown in Table 4. Based upon the results, it can be concluded that the proposed mechanism predicts well correlated results with the experimental data for the species profiles of H<sub>2</sub>O, CO<sub>2</sub>, and CO except the pressure profile, which could be due the reaction involvement in the formation of the species. The referenced mechanisms show the greater deviation with the experimental results.

#### 4. Conclusions

A major focus of this study was to develop a detailed kinetic mechanism to study the oxidation at in IC engine. The performance of this proposed reaction scheme was studied at both fuel lean ( $\phi < 1.0$ ) and rich conditions ( $\phi > 1.0$ ) with a range of fuel inlet condition of temperature ( $T_{ini} = 400-5,000^\circ\text{C}$ ) and pressure ( $P_{ini} = 1.0$  to 12 atm). Although discrepancies in various simulation profiles exist, the proposed kinetic mechanism showed good concordance with the GRI 3.0

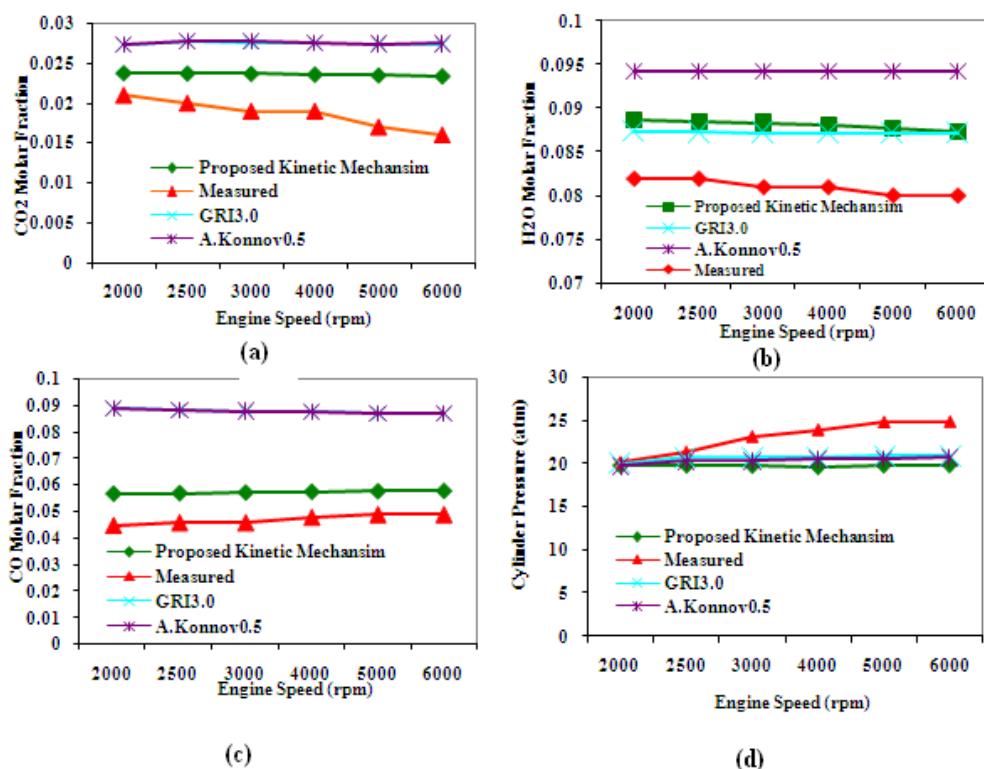


Figure 6. Effect of engine speed (rpm) on major product species fractions (CO<sub>2</sub>, H<sub>2</sub>O, CO) and cylinder pressure.

Table 4. Correlation coefficients of kinetic mechanisms with experimental data.

Mechanism Type	Correlation Coefficient			
	H <sub>2</sub> O	CO <sub>2</sub>	CO	Pressure
Proposed kinetic mechanism	0.9367	0.9753	0.9775	0.1616
GRI3.0	0.8817	0.1933	-0.963	0.8476
Aknnov 0.5	0.6324	0.1988	-0.9658	0.9192

mechanism. The experimental results especially in prediction of combustion pressure, temperature and major species profiles showed close agreement with the both reference mechanisms. Higher pollutants molar fractions (mean) are predicted using the proposed mechanism compared to both reference mechanisms (GRI 3.0 and Konnov 0.5). Based upon the simulation results and limited experimental data, the proposed kinetic model is a competitive candidate of natural gas combustion in IC engine.

### Acknowledgement

A special thank goes to Ms Warth from the Department of Chemical Physical Reactions, Nancy, France, for assisting in execution of EXGAS software, and we also acknowledge the Higher Education Commission (HEC) of Pakistan for financial assistance in the development of the experimental setup for this study.

### Nomenclature

T <sub>ini</sub>	= Initial Temperature [°C]
P <sub>ini</sub>	= Initial Pressure [atm]
CO	= Carbon monoxide
CO <sub>2</sub>	= Carbon dioxide
NO <sub>x</sub>	= Oxide of nitrogen (as NO, NO <sub>2</sub> , N <sub>2</sub> O)
k	= Reaction rate coefficient
A	= pre-exponential factor [mol-cm-sec-K]
T <sub>a</sub>	= Activation temperature [K]
b	= temperature exponent of pre-exponential factor
E	= activation energy of chemical reaction [cal/mole]
R	= ideal gas constant [J-mol <sup>-1</sup> K <sup>-1</sup> ]
Φ	= equivalence ratio

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